



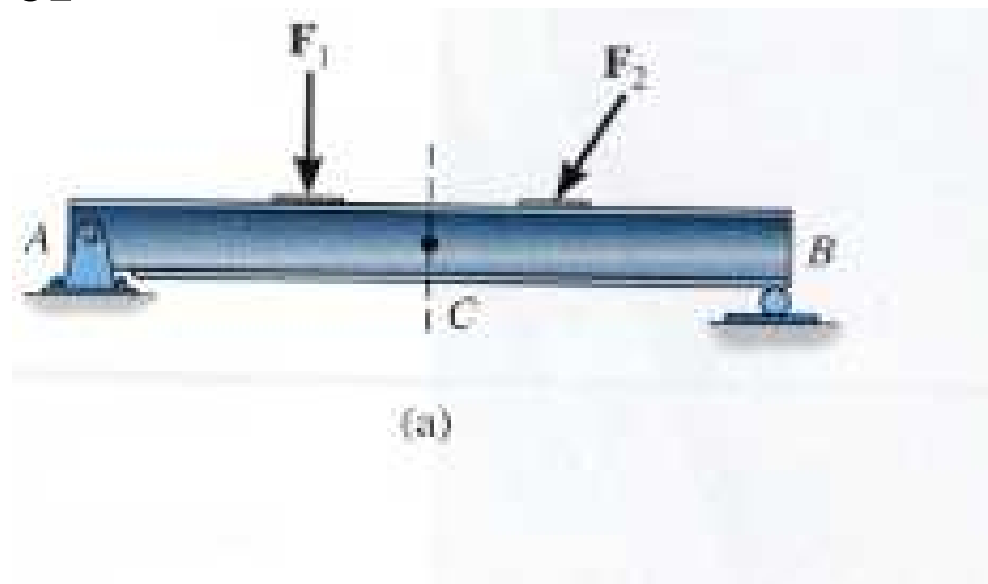
# Mechanics & Materials 1

## Chapter 7 Internal Loadings

FAMU-FSU College of Engineering  
Department of Mechanical Engineering

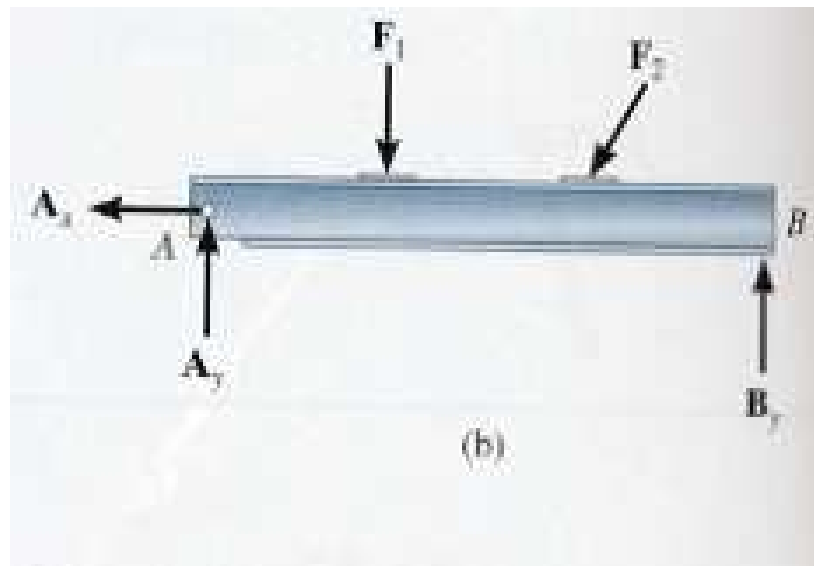
# Internal Loading

- Using the method of sections we can investigate the loading acting within a member



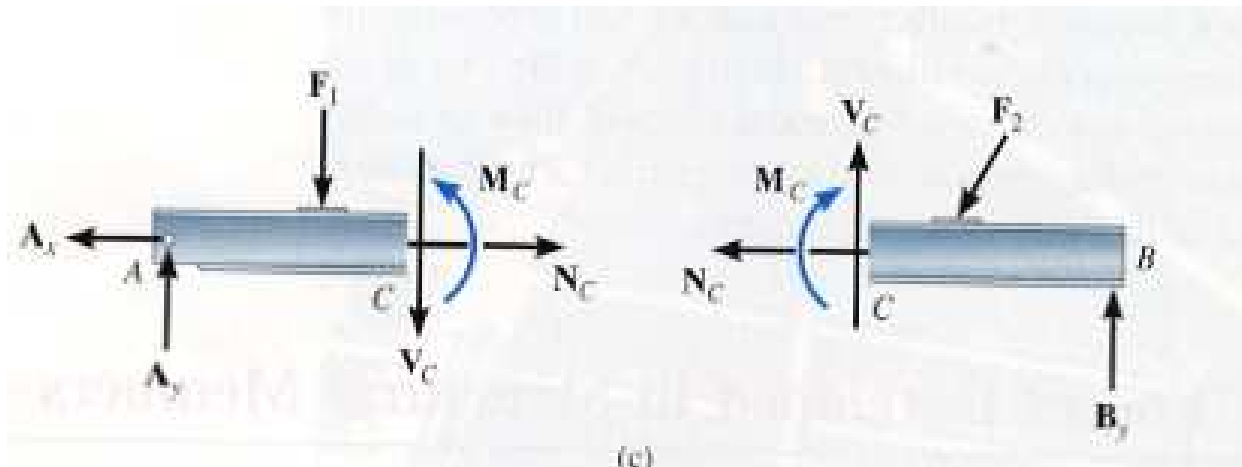
# Internal Loading

- Using a free body diagram of the ... beam we obtain the reactions  $A_x$ ,  $A_y$ ,  $B_x$ ,  $B_y$ .



# Internal Loading

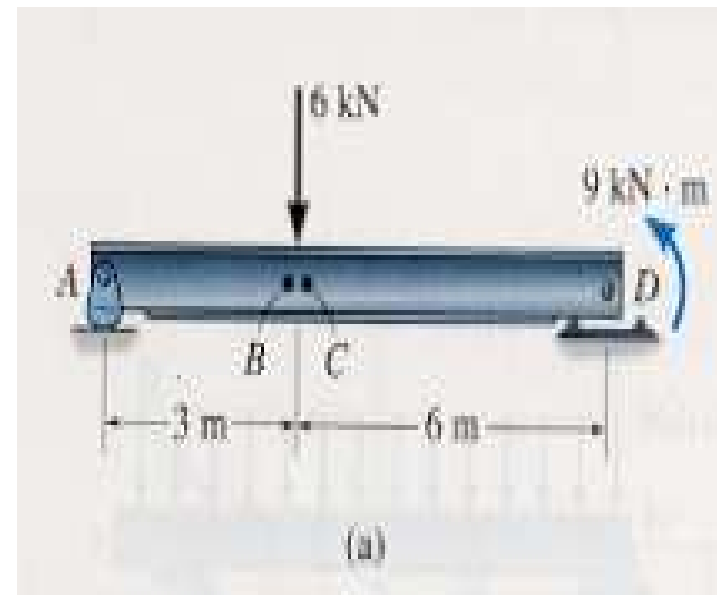
- Consider Section **AC** only, equilibrium of the sections requires adding three components
  - Force **N** along the axis of the beam is called axial loading
  - Force **V** perpendicular to the beam axis is called shear force
  - Moment **M** eliminates the moments of the forces **N** and **V**



# Internal Force: Analysis Procedure

## Example:

Determine the internal normal force, shear force, and the loading moment acting just to the left at point B, and just to the right at point C of the 6 kN force



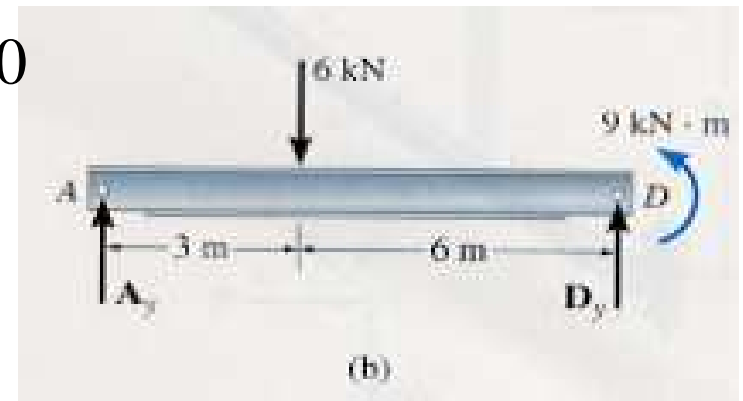
# Internal Force Analysis

## Step 1

- Draw a **free body diagram** of the entire frame and use it to determine as many of the reaction forces at the supports as you can.

$$\Sigma M_D = 0 \Rightarrow 9KN.m + (6KN)(6m) - A_y(9m) = 0$$

$$A_y = 5KN$$



# Internal Force Analysis

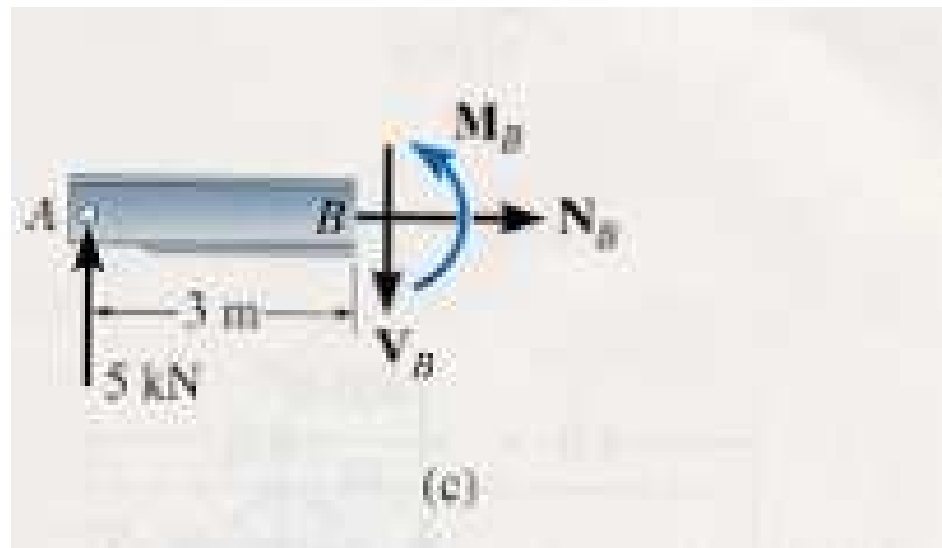
## Step 2

- Dismember the frame and draw a free body diagram of each of its members:
  - write as many equilibrium equations as are necessary to find all the forces acting on the member on which the point of interest is located.

# Internal Force Analysis

## Step 3

- Cut the member at a point of interest and draw a free body diagram of each of the two portions.





# Internal Loading Analysis

## Step 4

- Select one of the two free body diagrams you have drawn and use it to write three equilibrium equations for the corresponding point of the body

- Summing Moments about the point of interest

$$\sum M_p = 0 \rightarrow \text{gives bending moment at point P}$$

- Summing Forces along the axis of the member

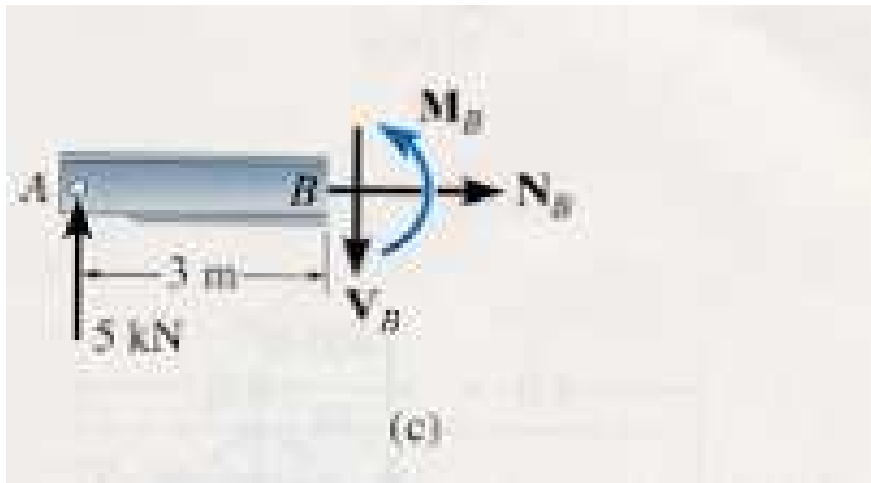
$$\sum F_x = 0 \rightarrow \text{will produce axial force in the member}$$

$$\sum F_y = 0 \rightarrow \text{shearing force}$$

# Internal Loading Analysis

## Step 4

### Segment AB



$$\xrightarrow{+} \sum F_x = 0$$

$$+\uparrow \sum F_y = 0$$

$$+\sum M_B = 0$$

$$N_B = 0$$

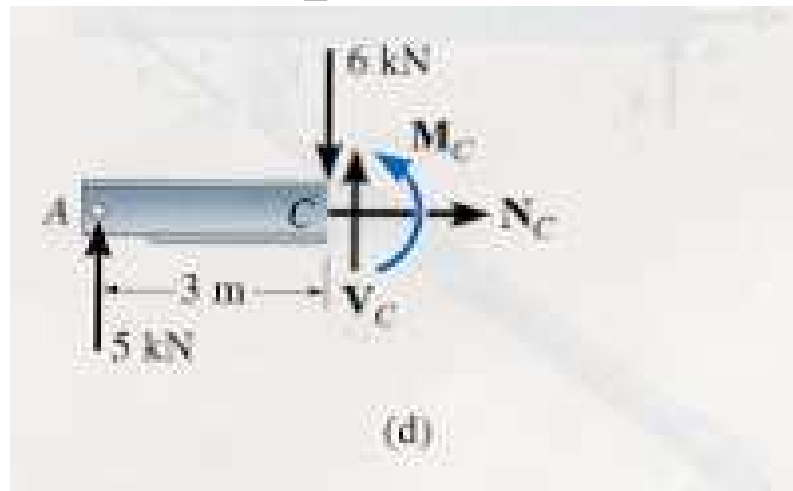
$$5kN - V_B = 0 \rightarrow V_B = 5kN$$

$$(-5kN)(3m) + M_B = 0 \rightarrow M_B = 15kN \cdot m$$

# Internal Loading Analysis

## Step 4

### Segment AC



$$\xrightarrow{+} \sum F_x = 0$$

$$+ \uparrow \sum F_y = 0$$

$$+ \sum M_C = 0$$

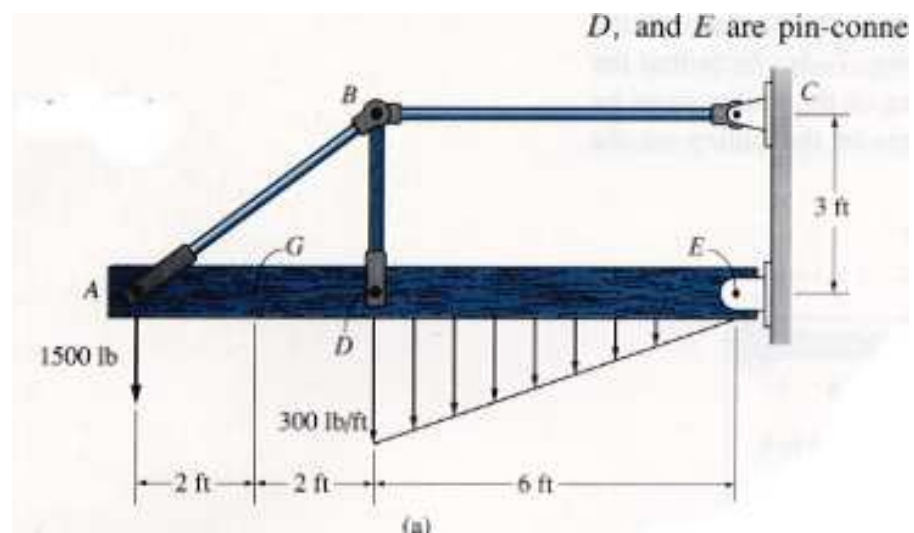
$$N_C = 0$$

$$5kN - 6kN + V_C = 0 \rightarrow V_C = 1kN$$

$$(-5kN)(3m) + M_C = 0 \rightarrow M_C = 15kN \cdot m$$

# Example

- Determine the resultant internal loading acting on the cross section at G of the wooden beam AE. Assuming the joints at A, B, C, D, and E are pin-connections



# Solution

## Step 1: Support Reaction

$$+\uparrow \sum F_y = 0$$

$$-1500 + \frac{1}{2}(6 \text{ ft})(300 \text{ lb/ft})$$

$$+ E_y = 0$$

$$E_y = 2400 \uparrow$$

$$+\sum M_E = 0$$

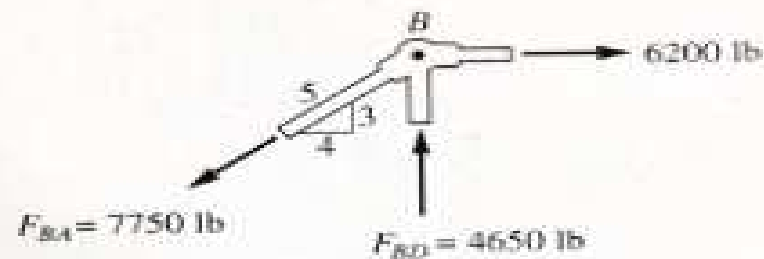
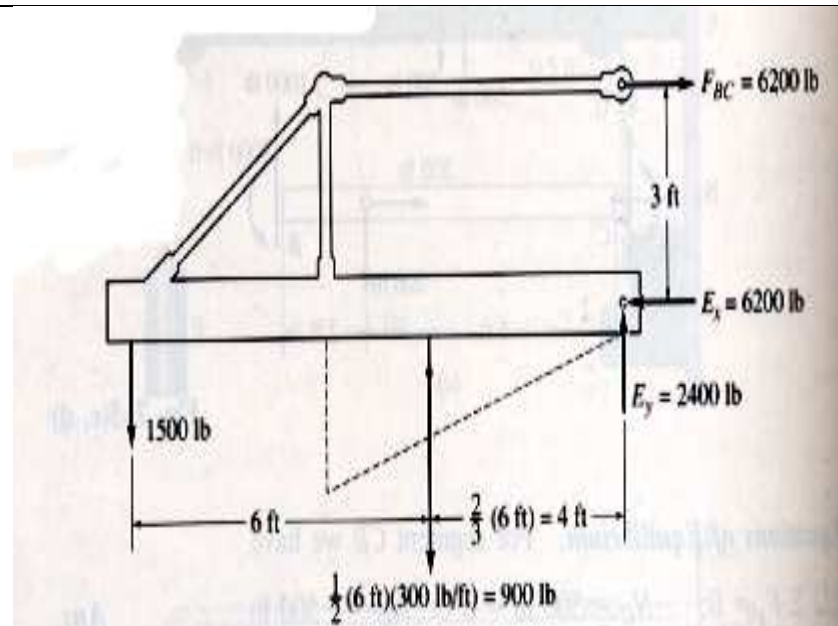
$$1500(10) + 900(4) - F_{C_x}(3) = 0$$

$$F_{C_x} = 6200 \text{ lb}$$

$$\longrightarrow \sum F_x = 0$$

$$-E_x + 6200 = 0$$

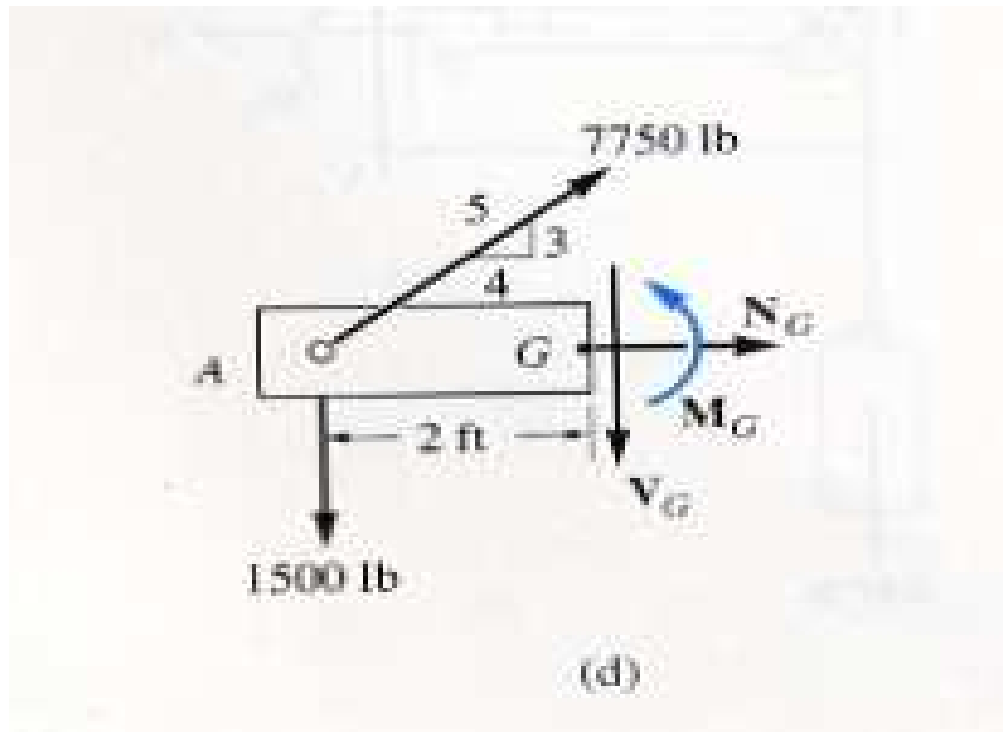
$$E_x = 6200$$



(c)

# Solution

Step 2 : Free Body Diagram at point G



# Solution

## Step 3 : Equations of Equilibrium for segment AG

$$\rightarrow \sum F_x = 0$$

$$7750\left(\frac{4}{5}\right) + N_G = 0$$

$$N_G = -6200\text{ lb}$$

$$+\uparrow \sum F_y = 0$$

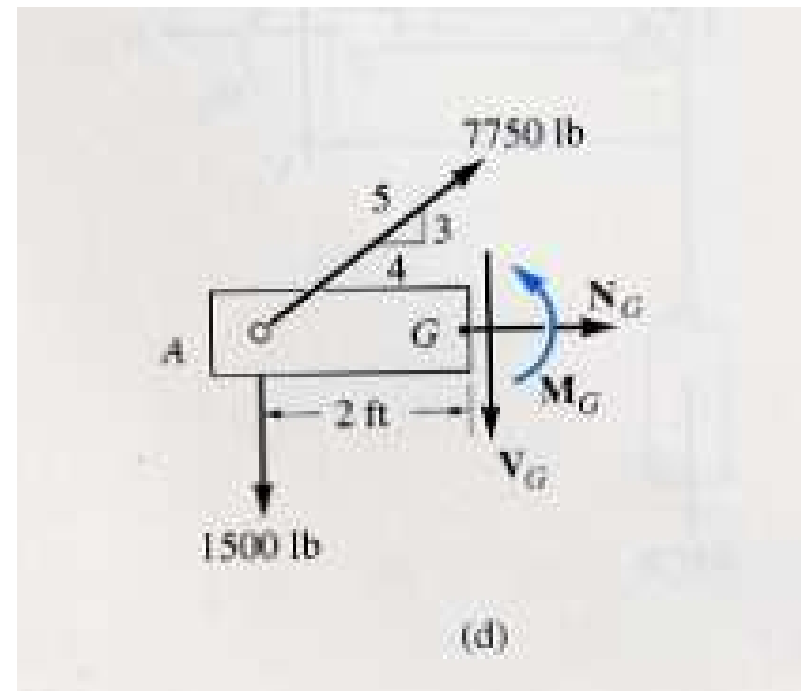
$$-1500 + 7750\left(\frac{3}{5}\right) - V_G = 0$$

$$V_G = 3150\text{ lb}$$

$$+\sum M_G = 0$$

$$M_G - 7750\left(\frac{3}{5}\right)(2) + 1500(2) = 0$$

$$M_G = 6300\text{ lb}\cdot\text{ft}$$



## Question ???

- Will we get the same results by considering segment GE rather than AG?

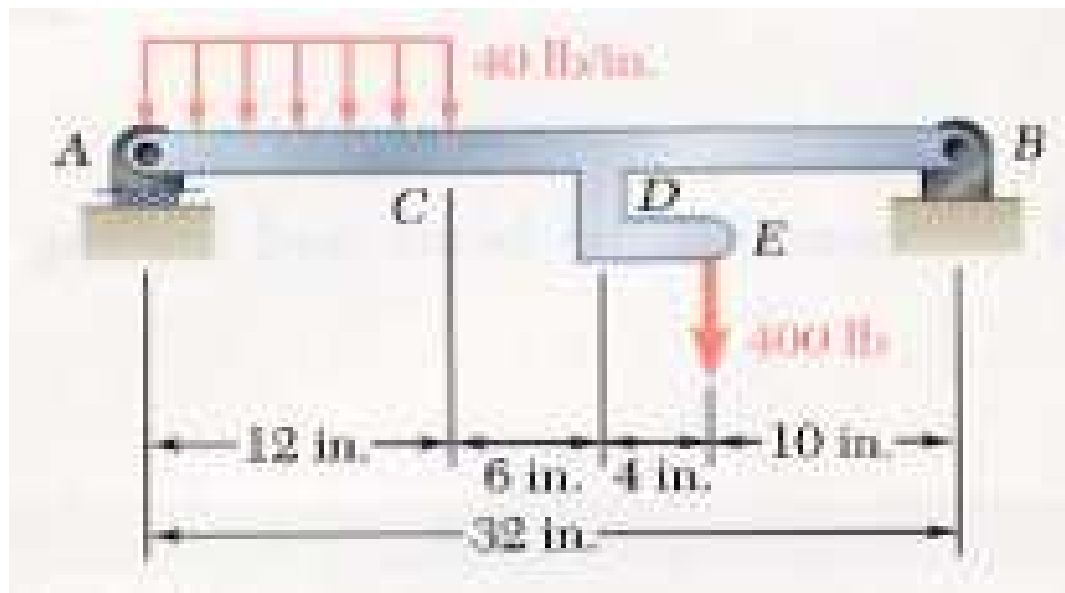


# Determining $V$ and $M$ in a Beam: General Procedures

- 1. Draw a **FBD** of the entire beam, find reactions and supports
- 2. Cut the beam at the point of interest
- 3. Draw FBD of the section of interest including :
  - the external loads and reactions
  - the shearing force and bending couple representing the internal forces
- 4. Write the **equilibrium equations** for the portion of the beam you selected
- 5. Record the values and signs of  $V$  and  $M$  obtained.

# Determining the Shear and Bending Moment in a Beam

- Draw the shear and bending moment diagrams for the beam shown below



# Step 1

- Draw the free body diagram of the entire beam, to determine the reactions at the beam supports

$$+\sum M_A = 0$$

$$B_y(32) - 480(6) - 700(22) = 0$$

$$B_y = 365 \text{ lb } \uparrow$$

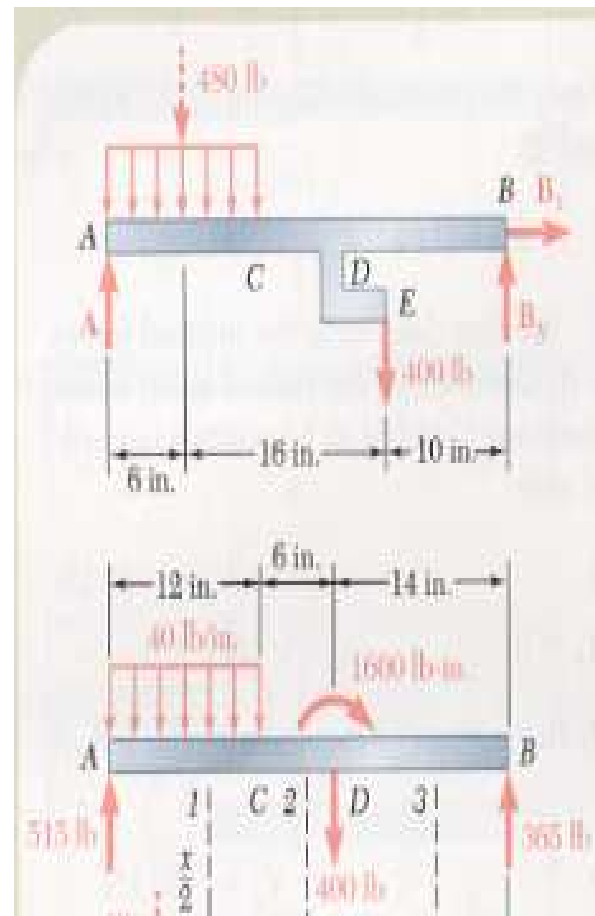
$$+\sum M_B = 0$$

$$480(26) + 400(10) - A(32) = 0$$

$$A = 515 \text{ lb } \uparrow$$

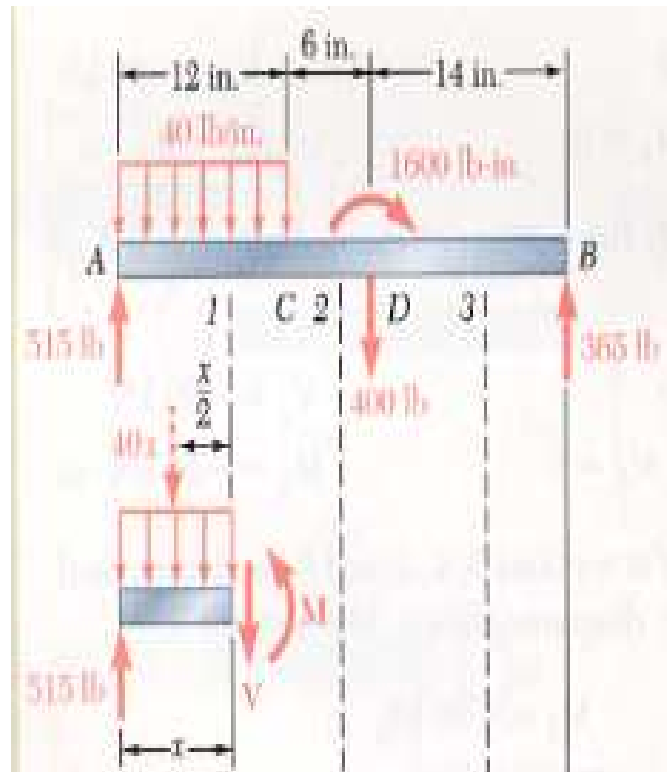
$$\rightarrow \sum F_x = 0$$

$$B_x = 0$$



## Step 2

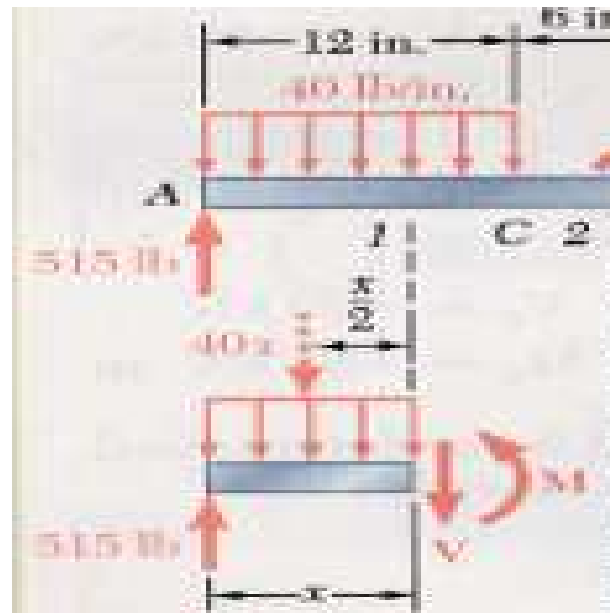
- Cut the beam at the point C ( point of interest)



## Step 3

- Draw free body diagram of the portion of the beam you have selected showing:
  - The loads and reactions exerted on that portion of the beam, **replace distributed loads by equivalent concentrated loads**
  - The shearing force and the bending moment representing the internal force at the point of interest

# Shear and Bending Moment from A to C



$$+ \uparrow \sum F_y = 0$$

$$+ \sum M_1 = 0$$

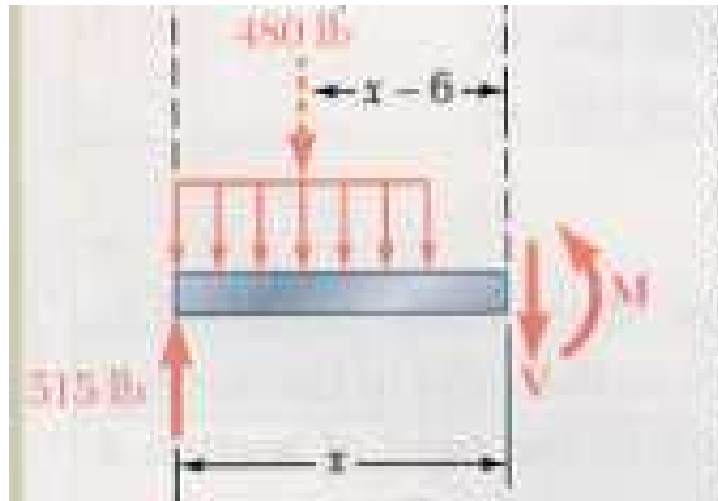
$$515 - 40(X) - V = 0$$

$$V = 515 - 40X \text{ lb}$$

$$- 515X - 40X \left( \frac{1}{2}X \right) + M = 0$$

$$M = 515X - 20X^2 \text{ lb} \cdot \text{in}$$

# Shear and Bending Moment from C to D



$$+ \uparrow \sum F_y = 0$$

$$+ \sum M_2 = 0$$

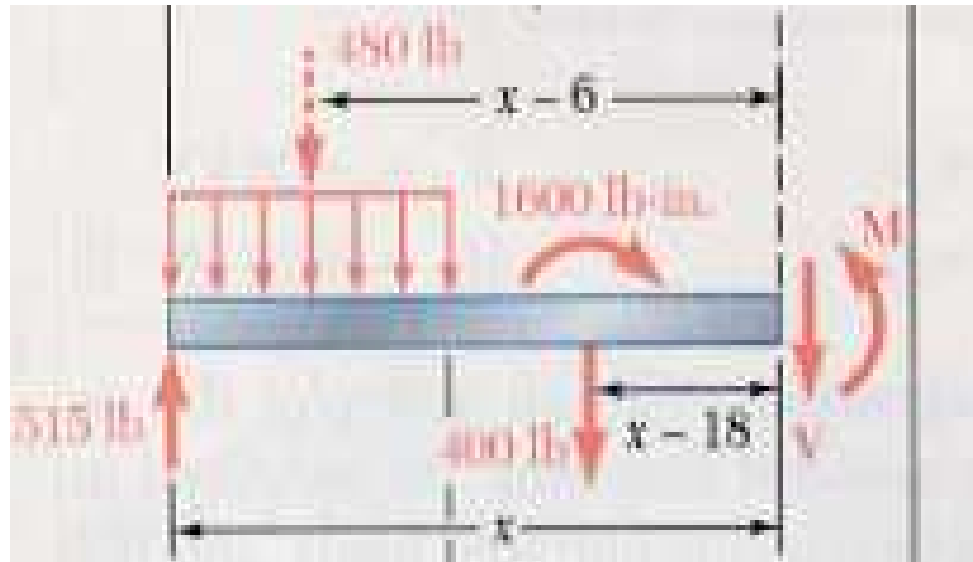
$$515 - 480 - V = 0$$

$$V = 35lb$$

$$-515X - 480(X - 6) + M = 0$$

$$M = (2880 + 35X)lb \cdot in$$

# Shear and Bending Moment from D to B



$$+ \uparrow \sum F_y = 0$$

$$+ \sum M_3 = 0$$

$$515 - 480 - 400V = 0$$

$$V = 365 \text{ lb}$$

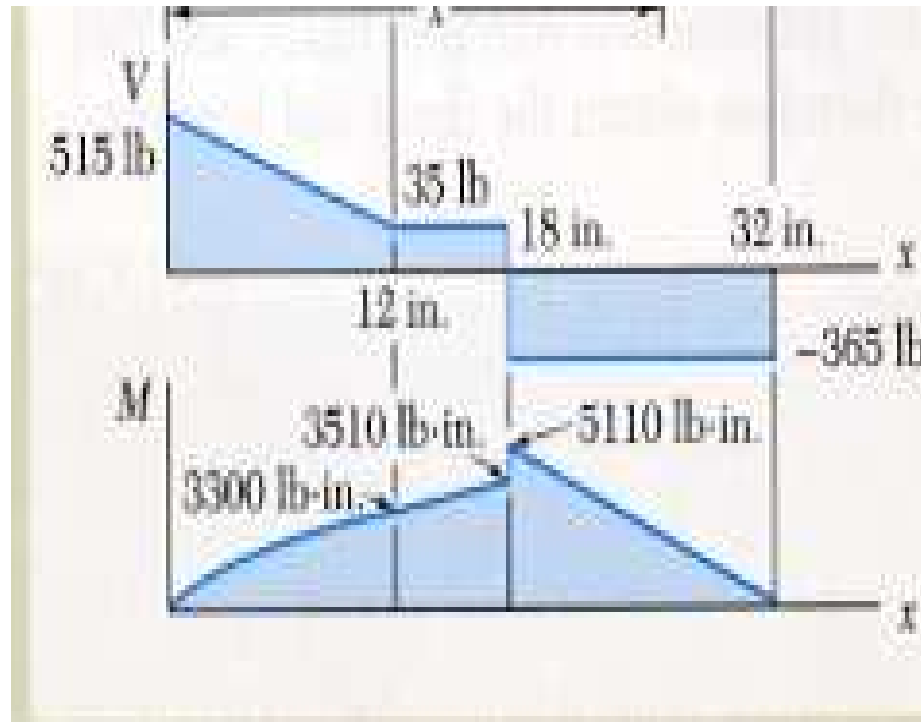
$$-515X - 480(X - 6) - 1600 + 400(x - 18) + M = 0$$

$$M = 11,680 - 365X \text{ lb} \cdot \text{in}$$



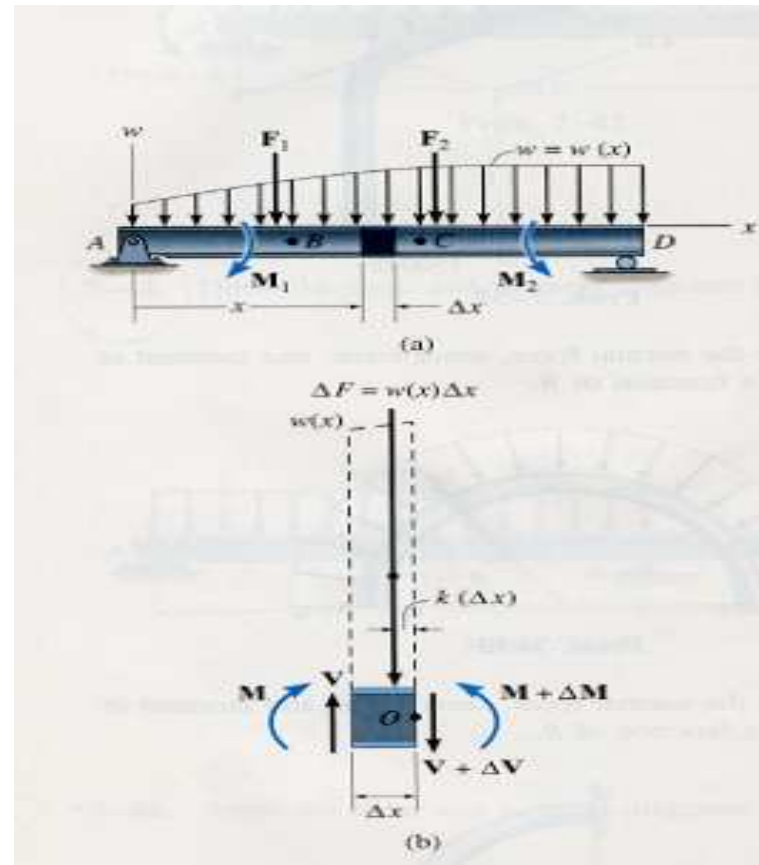
# Step 5

- Draw the shear and moment diagram

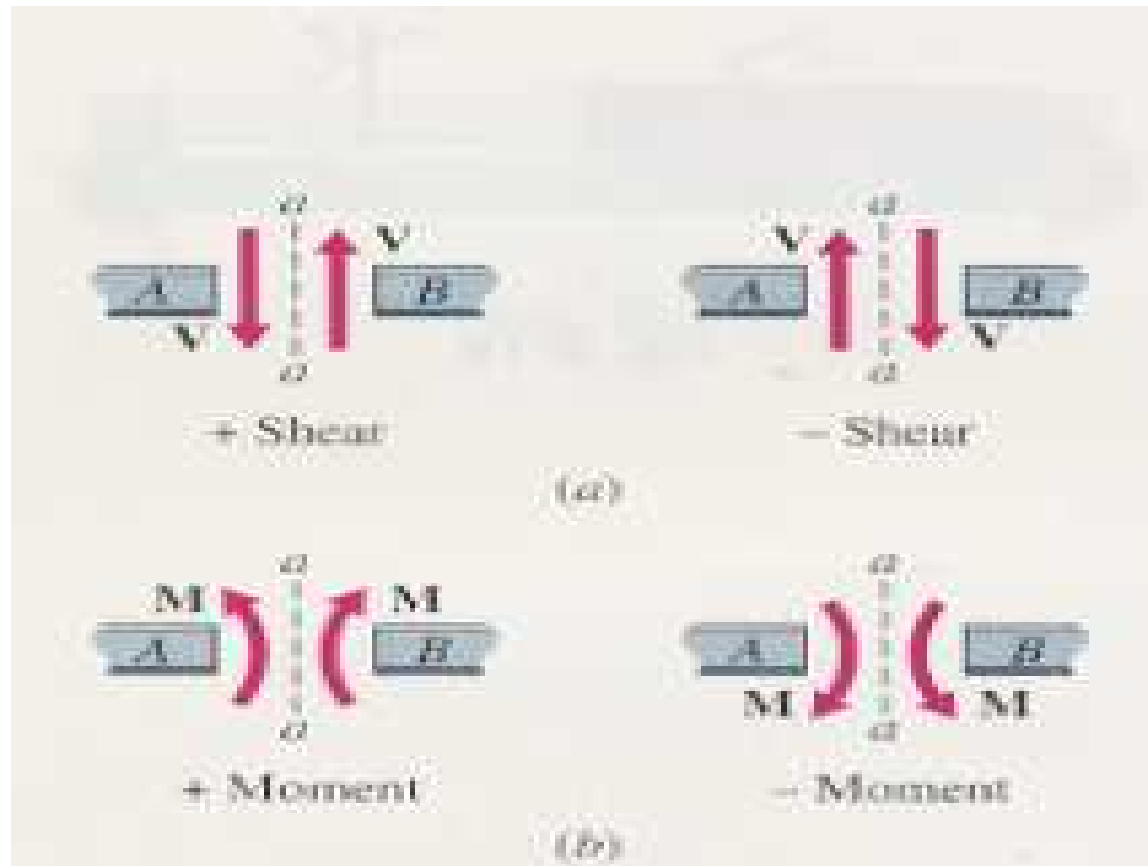


# Relationship Between Distributed Load, Shear, and Moment

- $w = w(x)$  is a distributed load
- Distributed load considered positive when acts downward, and negative when acts upward.
- Positive sense of shear force and bending moments are given



# Shear Force and Bending Moment Sign Convention



# w, V, M Relations

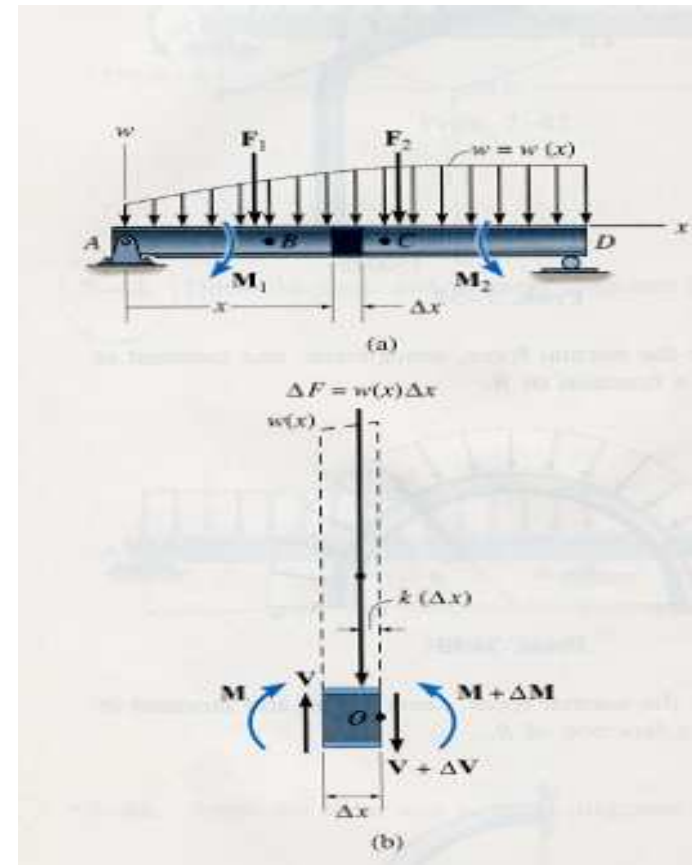
- Consider the equilibrium of the element cut from the beam:

$$+\uparrow \sum F_y = 0; \quad V - w(x) \Delta x - (V + \Delta V) = 0$$

$$\Delta V = -w(x) \Delta x$$

$$\downarrow + \sum M_O = 0; \quad -V \Delta x - M + w(x) \Delta x [k(\Delta x)] + (M + \Delta M) = 0$$

$$\Delta M = V \Delta x - w(x) k(\Delta x)^2$$



# Relation Between V&w

- Using the limits theory, dividing by  $\Delta x$  and taking the limit as  $\Delta x \rightarrow 0$

$$\frac{dV}{dx} = -w(x)$$

Slope of shear diagram = Negative of distributed load intensity

# Relation between M & V

$$\frac{dM}{dx} = V$$

Slope of  
moment diagram = Shear

# Area Method to Find V( Shear Force)

- *Change in shear between points B and C is equal to the negative of the area under the distributed - loading curve between these points*

$$\Delta V_{BC} = - \int w(x) dx$$

Change in shear = Negative of area under loading curve

# Area Method to Find M (Bending Moment)

- The change in moment between points B and C is equal to the area under the shear diagram within region BC*

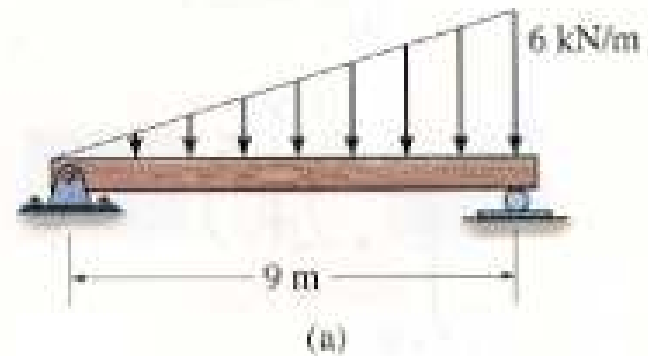
$$\Delta M_{BC} = \int V dx$$

Change  
in moment = Area under  
shear diagram



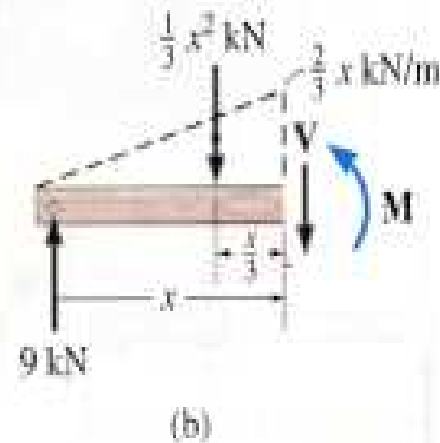
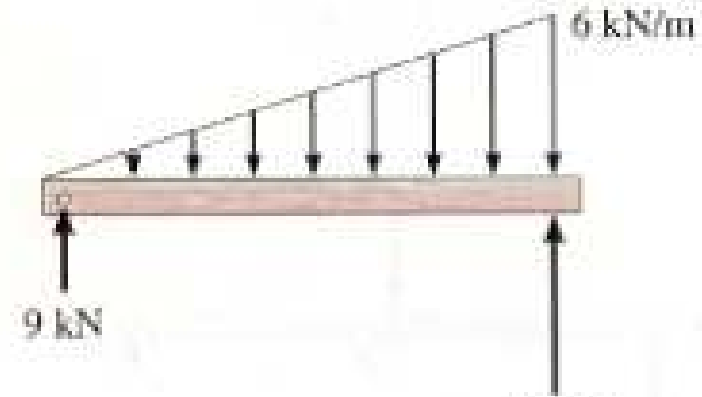
# Example

- Draw the shear and bending moment diagrams for the beam shown .



# Solution

- Step 1: FBD and reactions
- Step2: Cut a portion of the beam
- Step3: Draw a FBD of that section

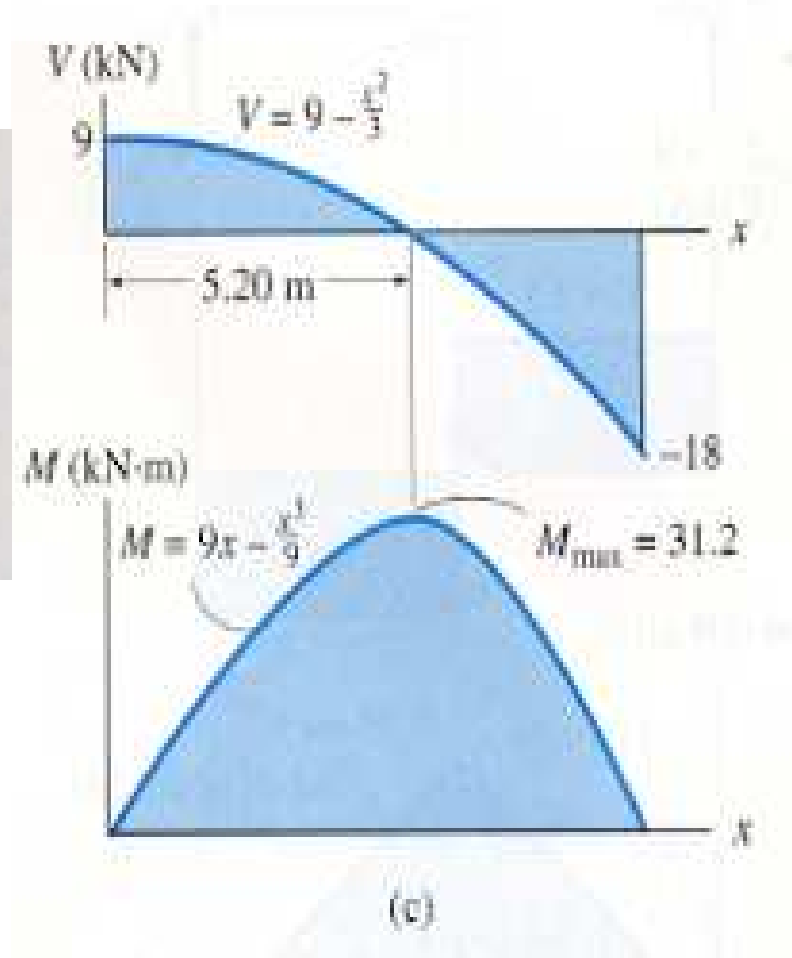


# Solution, Cont...
















- Step4: write the equilibrium equations for that section

$$\begin{aligned} + \uparrow \Sigma F_y = 0; & \quad 9 - \frac{1}{3}x^2 - V = 0 \\ & \quad V = \left(9 - \frac{x^2}{3}\right) \text{ kN} \\ \downarrow + \Sigma M = 0; & \quad M + \frac{1}{3}x^2\left(\frac{x}{3}\right) - 9x = 0 \\ & \quad M = \left(9x - \frac{x^3}{9}\right) \text{ kN} \cdot \text{m} \end{aligned}$$

- Step5: use the resulting equations to draw the V and M diagrams

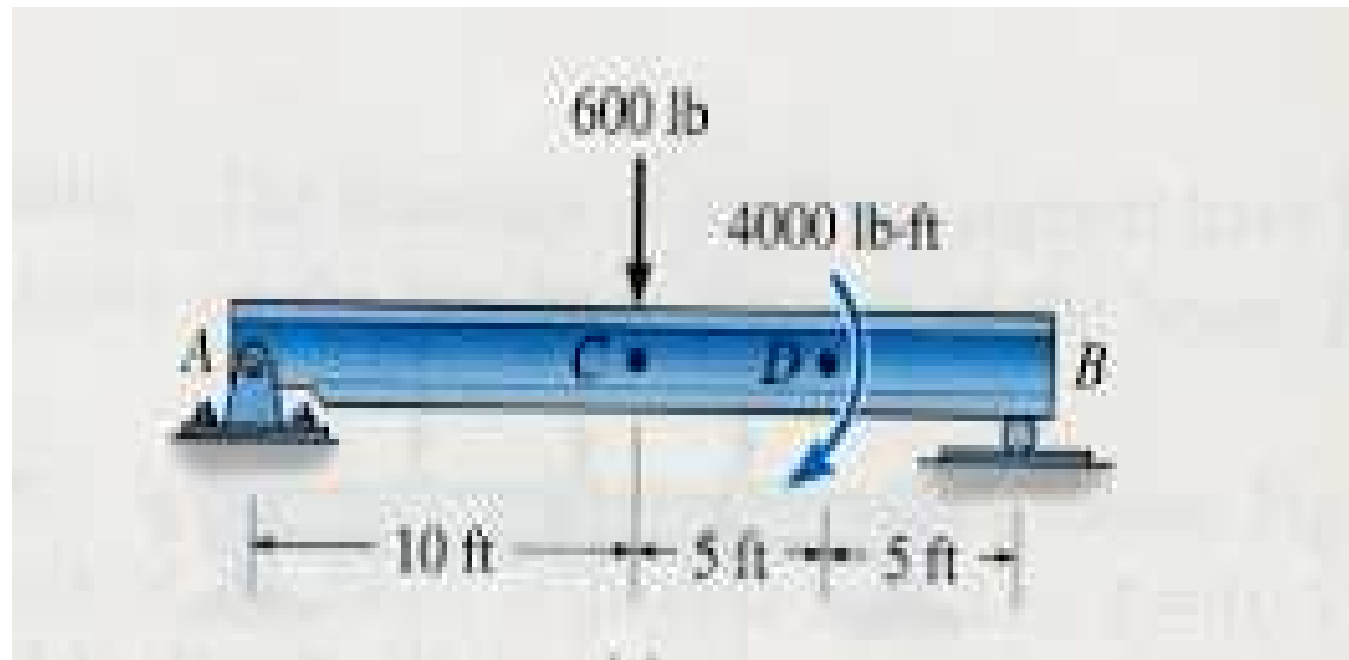


# Drawing V and M Diagrams Using the Area Method

Loading	Shear Diagram $\frac{dV}{dx} = -w$	Moment Diagram $\frac{dM}{dx} = V$
		
		
		
		
		

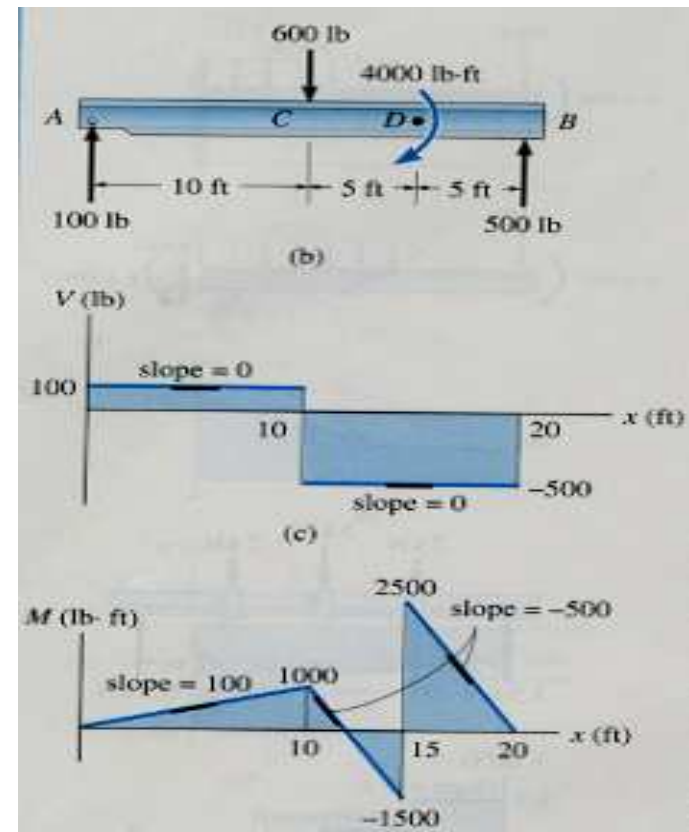
## Example : Area Method

- Using the area method draw the shear and moment diagrams of the beam shown below



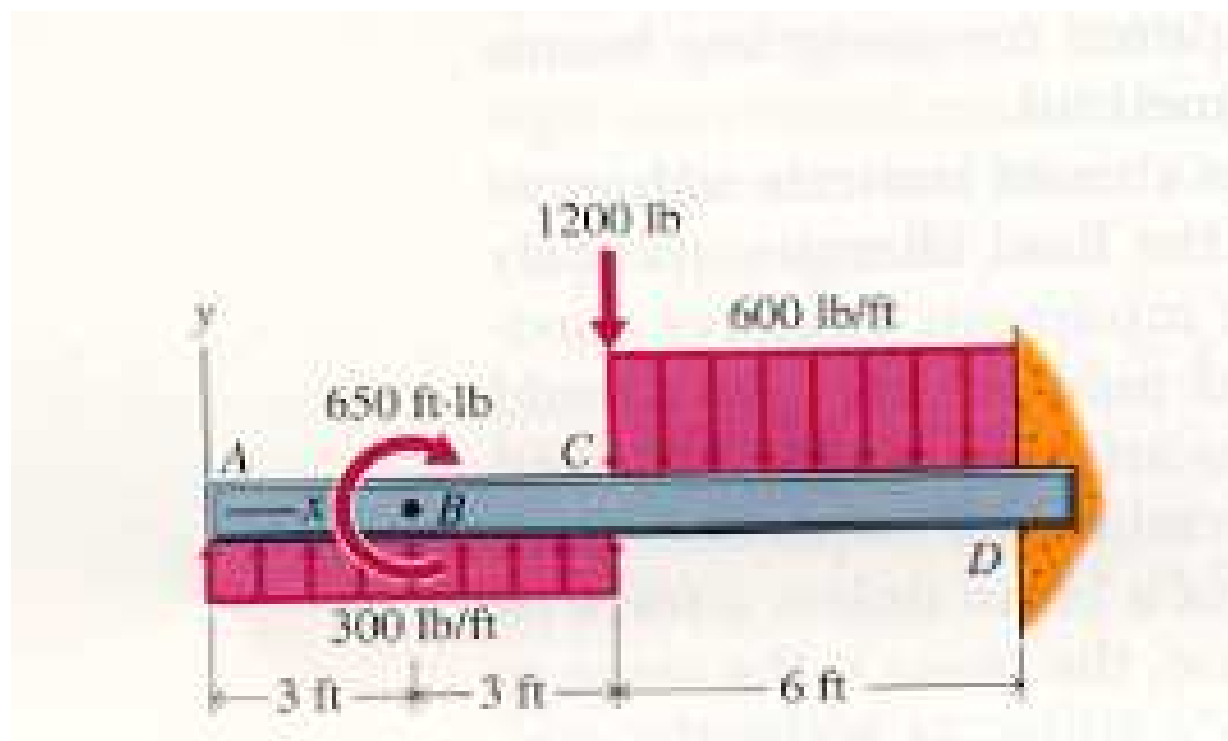
# Solution

- Note that when having a **concentrated couple** affecting the beam at certain point, that **will not affect the shear diagram**, however, the **bending moment diagram will show a discontinuity at that point** rising or falling by an amount equivalent to the magnitude of that couple

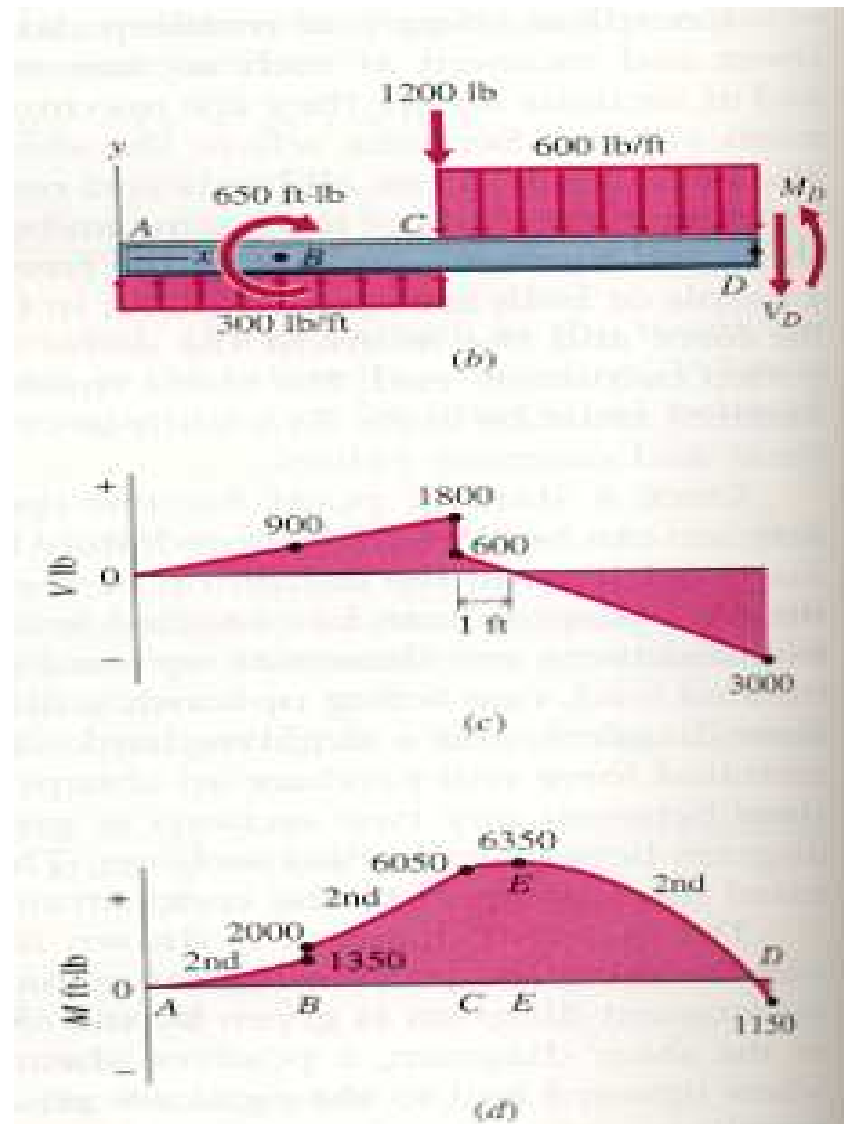


# Practice !

- Draw a complete shear and bending moment diagrams for the beam shown below



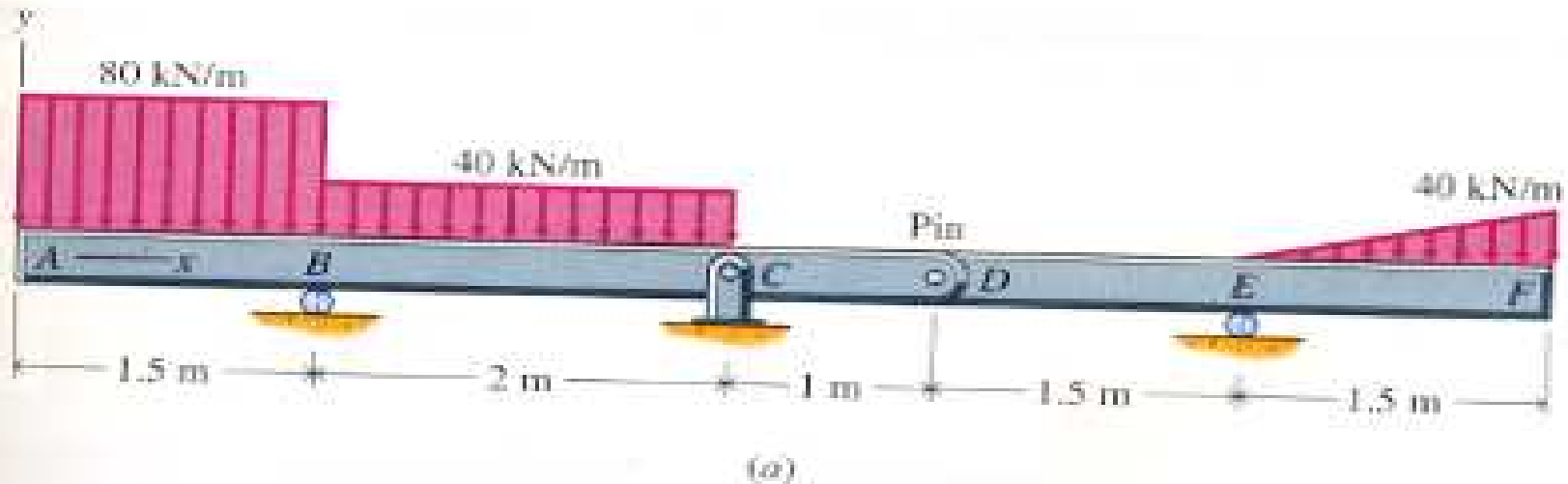
# Answer to Practice



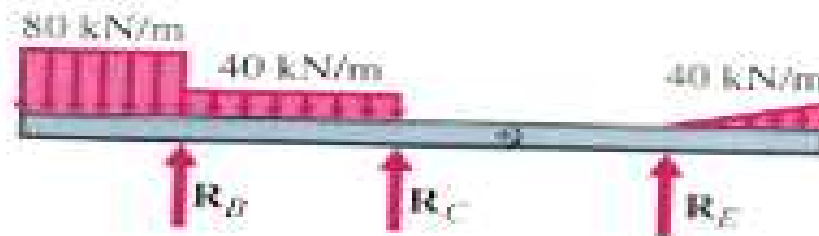


# Practice

- Draw the shear and bending moment diagrams for the following beam



# Answer to Practice



(b)



(c)



(d)