Typical Steady State Control Volume Problem Chart 2

(Not complete material coverage)

Inflow point (i)

Normally 4 variables are needed to fully determine it (since there is also an unknown velocity.)

2 known intensive variables (May get away with T only in some approximations, eg, when using saturated values as an approximation for compressed liquids.)



Tables, eg B.1.1-B.1.4; pv or Tv diagrams; $v = v_f + x(v_g - v_f)$ and similar for u, h, and

Ideal gas (applicable?) pv = RT (4 forms) Tables A5-A8 for u, h, s.

Make do with the formulae for *differences* in intensive variables listed below under "Device"?

Remaining intensive variables p, T, v, (x,)u, h, s.

Material flow: $w=\dot{W}/\dot{m}$ $q=\dot{Q}/\dot{m}$ $\dot{m}=\dot{V}/v=A\mathbf{V}/v$ $A=\frac{\pi}{4}D^2$

Device or Control Volume

C1: Type of device? Given that $\dot{Q} = 0$? Reversible? $\eta_{\text{turbine}} = w/w_s$; $\eta_{\text{compressor}} = w_s/w_s$

C2: Mass:
$$\sum \dot{m}_i = \sum \dot{m}_e$$
 Energy:
$$\dot{Q} + \sum \dot{m}_i \left(h_i + \frac{1}{2} \mathbf{V}_i^2 ? + g Z_i ? \right) = \sum \dot{m}_e \left(h_e + \frac{1}{2} \mathbf{V}_e^2 ? + g Z_e ? \right) + \dot{W}$$

C3:
$$\dot{W} = 0$$
? Or $w + \Delta KE$? $+ \Delta PE$? $= 0$ $\left| -v\left(p_e - p_i\right) \right| \left| \frac{n\left(p_e v_e - p_i v_i\right)}{1 - n} \right| - pv \ln\left(\frac{p_e}{p_i}\right)$? $q = [0 \text{ and } s_2 = s_1]$ $\left| T(s_2 - s_1) \right| \text{ other}$? $\sum \dot{m}_e s_e - \sum \dot{m}_i s_i = \sum \frac{\dot{Q}}{T} + \dot{S}_{\text{gen}}$

For ideal gasses:

$$\begin{split} h_2 - h_1 &= \int_1^2 C_p \; \mathrm{d} T \approx C_{p_{\mathrm{ave}}}(T_2 - T_1) \\ s_2 - s_1 &= s_T^0(T_2) - s_T^0(T_1) - R \ln \left(\frac{p_2}{p_1}\right) \approx C_{p_{\mathrm{ave}}} \ln \left(\frac{T_2}{T_1}\right) - R \ln \left(\frac{p_2}{p_1}\right) = \dots \\ \text{Polytropic: } \frac{p_2}{p_1} &= \left(\frac{v_1}{v_2}\right)^n = \left(\frac{T_2}{T_1}\right)^{\frac{n}{n-1}} & \text{isothermal: } n = 1 \\ \text{isentropic and } k \text{ constant: } n = k \end{split}$$

For compressed liquids, by approximation, best at constant pressure:

$$h_2 - h_1 \approx C_{(p)_{\text{ave}}} (T_2 - T_1)$$
 $s_2 - s_1 \approx C_{(p)_{\text{ave}}} \ln \left(\frac{T_2}{T_1} \right)$

Exit point (e)